

## Chapter 1

### Mechanisms and Machines: Basic Concepts

1.1 a. The ball-joint (spherical pair) has three degrees-of-freedom, the prismatic pair one, and the cylindrical pair two. The number of degrees-of-freedom of the spatial linkage is given by

$$\begin{aligned}
 DF_{\langle \text{spatial} \rangle} &= 6(n_L - n_J - 1) + \sum f_i \\
 &= 6(4 - 3 - 1) + 3 + 1 + 2 \\
 &= 6 \text{ degrees-of-freedom}
 \end{aligned}$$

(We do not need the inequality sign for this open-loop chain).

b. Treating the construction equipment schematic as a spatial linkage:

$$\begin{aligned}
 DF_{\langle \text{spatial} \rangle} &\geq 6(n_L - n_J - 1) + \sum f_i \\
 &\geq 6(9 - 11 - 1) + 9 + 2 \times 2
 \end{aligned}$$

$$DF_{\langle \text{spatial} \rangle} \geq -5$$

c. Treating the construction equipment schematic as a planar linkage, where a sliding pair has only one degree-of-freedom in plane motion:

$$\begin{aligned}
 DF_{\langle \text{planar} \rangle} &= 3(n_L - n_J - 1) + \sum f_i \\
 DF_{\langle \text{planar} \rangle} &= 3(9 - 11 - 1) + 9 + 2 \\
 DF_{\langle \text{planar} \rangle} &= 2
 \end{aligned}$$

d. The planar motion assumption applies if motion occurs in a plane or in a set of parallel planes. The planes of motion of the links must all be parallel. The revolute joint axes must be perpendicular to those planes. These conditions do apply to the construction machinery. The operator controls the linkage via the two hydraulic cylinders.

1.1(a) For the linkage as sketched originally,

$$DF = 3(n_L - 1) - 2n_J = 3(9 - 1) - 2 \times 12 = 0$$

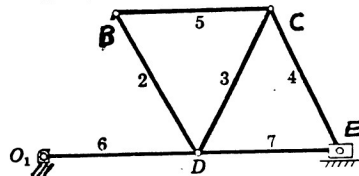
a) Removing any single link (one through seven) reduces  $n_L$  by one and  $n_J$  by two from which

$$DF = 3(8 - 1) - 2 \times 10 = 1.$$

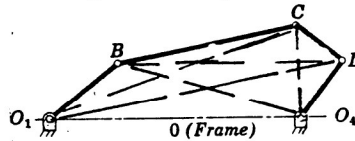
b) Also, we obtain  $DF = 1$  by removing links 1, 2 and 5; or 3, 4 and 5; or 1, 2, 3, 4, and 5.

c)  $DF = 1$  if all links are removed except link 1, or 6 or the slider.

d)  $DF = 1$  if the slider is removed, but the joint at E retained.



1.2(b) Add one link extending from  $O_1$  to C, or B to D, or C to  $O_4$ , or B to  $O_4$ , or  $O_1$  to D. Then,  $DF = 3(n_L - 1) - 2n_J' = 3(6 - 1) - 2 \times 7 = 1.$



1.3 Stroke length  $S = 2R.$

$$v(\text{avg}) = 2Sn = 2(4 \text{ in})(3000 \text{ rev/min}) = 24,000 \text{ in/min or } 400 \text{ in/sec}.$$

$$1.4 R = 2, L = 4, E = 1 \text{ in. } \omega = \frac{2\pi}{60} (3000) = 314 \text{ rad/sec}$$

Referring to fig. 1.16

$$\phi_1 = \arcsin E/(L-R) = 30^\circ.$$

$$\phi_2 = \arcsin E/(L+R) = 9.6^\circ.$$

$$\alpha = 180^\circ + \phi_1 - \phi_2 = 200.4^\circ \text{ or } 3.50 \text{ rad.}$$

$$\beta = 180^\circ - \phi_1 + \phi_2 = 159.6^\circ \text{ or } 2.79 \text{ rad.}$$

$$\text{Stroke } S = [(L+R)^2 - E^2]^{1/2} - [(L-R)^2 - E^2]^{1/2} = 4.184 \text{ in.}$$

$$\text{"Forward" stroke time } t_1 = \alpha/\omega = 0.01114 \text{ sec. } v_1(\text{avg}) = S/t_1 = 375.5 \text{ in/sec.}$$

$$\text{"Return" stroke time } t_2 = \beta/\omega = 0.00889 \text{ sec. } v_2(\text{avg}) = S/t_2 = 470.6 \text{ in/sec.}$$

1.5 (see 1.4)  $\phi_1 = 48.5^\circ$ ,  $\phi_2 = 14.5^\circ$ ,  $\alpha = 3.73$  rad,  $\beta = 2.55$  rad.

$S = 4.49$  in,  $t_1 = 0.01187$  sec,  $v_1(\text{avg}) = 378$  in/sec.

$t_2 = 0.00812$  sec,  $v_2(\text{avg}) = 553$  in/sec.

1.6  $\omega = 3000 \pi/30 = 314.16$  rad/s

$$\phi_1 = \arcsin (E/(L-R))$$

$$= \arcsin (50/(200-100)) = .5236 \text{ rad}$$

$$\phi_2 = \arcsin (E/(L+R))$$

$$= \arcsin (50/(200+100)) = .1674 \text{ rad}$$

$$\alpha = \pi + \phi_1 - \phi_2 = 3.4977 \text{ rad} \quad \beta = \pi - \phi_1 + \phi_2 = 2.7854 \text{ rad}$$

Forward stroke time  $t_1 = \alpha/\omega = .01113$  s.

Return stroke time  $t_2 = \beta/\omega = .008866$  s.

$$\text{Stroke } S = \sqrt{(L+R)^2 - E^2} - \sqrt{(L-R)^2 - E^2} = 209.20 \text{ mm}$$

Avg. vel. forward  $v_1(\text{avg}) = S/t_1 = 18790$  mm/s

Avg. vel. return  $v_2(\text{avg}) = S/t_2 = 23595$  mm/s

1.7 The crank and connecting rod positions are determined as in the flow chart. The linkage skeleton diagram is shown on the sketch for  $L/R=1.5$ ,  $E/R=0.2$  where crank angle  $T_1$  varies from  $0^\circ$  to  $340^\circ$  in  $20^\circ$  steps.

Angles  $T_1$  and  $T_2$  and slider location  $X_2$  are tabulated below for  $R=1$ :

$T_1$	$T_2$	$X_2$
0	-7.66225566877	2.48660687473
20	5.43290764513	2.43295424518
40	17.1690352815	2.19920149468
60	26.3604597467	1.84402758957
80	31.5474944957	1.45195830359
100	31.5474944957	1.12466185325
120	26.3604597467	0.844227589566
140	17.1690352815	0.667112607638
160	5.43290764513	0.553569803629
180	-7.66225566877	0.486606874732
200	-21.1829283574	0.438954641475
220	-34.1844166885	0.474805686816
240	-45.2905623587	0.555267662011
260	-52.1735359216	0.746259746205
280	-52.1735359216	1.09355610154
300	-45.2905623587	1.56526766201
320	-34.1844166885	2.00689457325
340	-21.1829283574	2.38833983205
360	-7.66225566877	2.48660687473